A magnetic falling-sphere viscometer

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ABSTRACT

We present a falling-sphere viscometer with a magnetized sphere and fluxgate magnetometers continuously measuring the magnetic field produced at the sensor positions by the falling sphere. With a fluid volume of 15 ml and within a few seconds, we directly measure dynamic viscosities in a range between 200 and 3000 cP with a precision of 3%.

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I. INTRODUCTION

The measurement of the viscosity of (Newtonian) fluids finds applications in several industries, such as the pharmaceutical, food, cosmetic, and lubricant industries. Based on their operating principles, viscometers can be roughly divided into (i) mechanical, (ii) microfluidic, and (iii) electromagnetic. Early mechanical viscometers still in use are capillary viscometers, where viscosity is measured by timing the fluid flow through a narrow capillary. Another type of mechanical viscometer measures the torque required to rotate a body (e.g., a disk or a cylinder) inside the fluid. Yet another mechanical viscometer is the falling-sphere viscometer, where the viscosity is found by measuring the terminal velocity of a sphere falling through the fluid under gravity, friction, and buoyancy. Modern microfluidic technology has led to compact devices requiring a small fluid sample volume. Finally, what we term electromagnetic viscometers are devices using some electromagnetic effect coupled to viscous flow. For example, a ferrofluid viscometer measures the relaxation of a magnetized ferrofluid in the sample under consideration.

We here introduce a falling-sphere viscometer with a "magnetic twist." We use a magnetized sphere and fluxgate magnetometers continuously reading the changing magnetic field produced by the falling sphere at the position of the sensors. By fitting the fluxgate signals to a theoretical form, we can extract the fluid’s viscosity with a precision of 3%. Our viscometer is rather compact (volume occupied by sensors and the sample is about 5 x 5 x 10 cm³), the measurement time is a few seconds, and the required fluid volume is less than 15 ml. It is worth noting that we directly access the dynamic viscosity of the fluid. In contrast, conventional falling-sphere viscometers measure the sphere’s terminal velocity, which depends both on the dynamic viscosity and on the fluid’s mass density. Compared to other falling-sphere viscometers, our viscometer is similar to the optical designs using a camera to monitor the sphere’s fall in that they both use some physical means (optical vs magnetic) to track the falling ball. While optical viscometers require transparent fluids, such optical designs report a higher accuracy than the one arrived in this work at an expense of a more elaborate apparatus. One further difference could be cost; however, a direct comparison is not meaningful as the technology and cost of cameras vs fluxgate sensors is changing rapidly.

In Sec. II, we provide a theoretical description of the experiment presented in Sec. III. In Sec. IV, we analyze the measurement results and errors, while in Sec. V, we elaborate on several possible sources of measurement uncertainty. In the conclusions of Sec. VI, we discuss some possibilities for further developing this methodology.

II. THEORETICAL DESCRIPTION

Consider a sphere of mass $m$, radius $r$, and mass density $\rho_s = m/\frac{4}{3}\pi r^3$, moving in a fluid of dynamic viscosity $\eta$. The conventional falling-sphere viscometer measures the sphere’s terminal velocity in the fluid, $v_{\text{term}}$, under the action of (i) the gravitational force $F_g = mg$, (ii) the Stokes frictional force $F_S = 6\pi r\eta v$, and (iii) the buoyant force $F_B = \frac{1}{2}\pi r^2\rho_f g$, where $\rho_f$ is the fluid’s mass density.
Once the falling sphere reaches the terminal velocity under a force equilibrium, it will be \( F_g = F_m + F_b \), from which the equation follows that \( \eta = 2\pi^2(\rho_f - \rho_d)g / 9v_m \). The terminal velocity is measured by timing the sphere as it traverses a known distance. Given the fluid’s density, the viscosity can be found.

The viscometer presented here does not rely on the measurement of \( v_m \), but on the whole trajectory of the sphere from the top of the fluid column to its bottom, described by the sphere’s height as a function of time, \( z(t) \). Initially, a neodymium sphere is held at rest by a current-carrying coil, just above the fluid column’s top surface at height \( z = H \), as shown in Fig. 1(a). When the current is switched off at \( t = 0 \), the sphere commences its fall within the fluid. The coordinate system, as shown in Fig. 1(a), has the coordinate center at the bottom and center of the cylindrical fluid column.

The height of the sphere can be found by solving the equation of motion \( m\ddot{z} = -mg - 6\pi\eta r \dot{z} + F_b \), with \( F_b \) as given before. The initial conditions are \( z(0) = H \) and \( \dot{z}(0) = 0 \). Defining the time constant \( \tau = 2\rho_f r^2 / 9\eta \), it follows that

\[
\dot{z}(t) = H + g \left( \frac{1 - \rho_f}{\rho_d} \right) \tau^2 \left( 1 - \frac{t}{\tau} - e^{-t/\tau} \right). \tag{1}
\]

The time constant \( \tau \) quantifies the time it takes for the sphere to reach terminal velocity, i.e., when the exponential term in Eq. (1) has become negligible. We stress that our measurement does not rely on the sphere reaching a terminal velocity; i.e., when analyzing data measured time-dependent magnetic fields to a theoretical form derived herein.

A. Measurable viscosity range

The SI unit of viscosity is 1 Ns/m² = 1000 cP. For example, the viscosity of engine oil at room temperature is about 500 cP. Given the sphere’s density \( \rho_f \approx 7.47 \) g/cm³, it follows that the corresponding value of the parameter \( \tau \) is 4 ms. The sphere’s density was estimated from the mass and radius data given by the manufacturer for much larger spheres of the same material, in order to minimize the relative error of the estimate.

To find the range of values of \( \tau \) measurable with our methodology, we first note that, as is evident from Eq. (1) by expanding the exponential term to second order, for \( t \ll \tau \), the height of the sphere \( z(t) \) ceases to depend on \( \tau \). Thus, a small viscosity (large \( \tau \)) is not measurable using a too short trajectory since the sphere will practically undergo free fall at early times. For example, the time to reach the bottom of our 9.5 cm cylinder by free fall is about 0.15 s; hence, this would be an approximate upper limit for the measurability of the parameter \( \tau \) with such a device, translating into a lower limit for the viscosity of \( \eta \approx 10^{-20} \) cP for typical fluid densities. The upper limit of the measurable viscosity can, in principle, be arbitrarily high, as long as the sphere does not fall through the fluid.

B. Magnetometer signals

The magnetic field produced by a magnetic dipole of moment \( \mathbf{m} \) at the position vector \( \mathbf{r} \) with respect to the dipole is \( \mathbf{B} = \mu_0 / 4\pi \frac{3\mathbf{m} \cdot \mathbf{r} \mathbf{r} - \mathbf{m} \mathbf{r}}{r^5} \), where \( r = |\mathbf{r}| \). As shown in Fig. 1, we use two fluxgate sensors adjacent to the fluid, with their sensitive axes being along the z axis, the sphere’s trajectory. For the moment, we consider point sensors, and later, we will take into account the finite sensing volume. Let the position of the jth fluxgate sensor be denoted by the position vector \( (a_j, b_j, c_j) \), where \( j = 1, 2 \). That is, we consider the two point sensors to define a line parallel to the z axis. Then, the position of the jth sensor with respect to the falling sphere is \( \mathbf{r}_j = a_j \mathbf{x} + b_j \mathbf{y} + (c_j - z(t)) \mathbf{z} \). Thus, the signal of the jth sensor will be \( B_j(t) = \mathbf{z} \cdot \mathbf{B} \).

At time \( t = 0 \), the magnetization of the sphere is aligned with the axis of the current-carrying coil, the z axis. Setting \( \mathbf{m} = m\hat{z} \) and \( B_0 = \mu_0 m / 4\pi \), we find

\[
B_j(t) = B_0 \frac{1}{(a_j^2 + b_j^2)^{3/2}} \frac{2f_j^2(t) - 1}{[1 + f_j^2(t)]^{3/2}} + b_j, \tag{2}
\]
where

\[ f(t) = \frac{c_j - z(t)}{\sqrt{x^2 + b^2}}, \quad (3) \]

and \( b_0 \) being a background magnetic field common to both sensors. By measuring the difference \( \Delta B = B_1(t) - B_2(t) \), the background field drops out. This helps suppress common magnetic fields, in particular, ac magnetic fields from nearby 50 Hz power lines. In summary, the viscosity \( \eta \) hides in the parameter \( r \) entering the sphere’s height \( z(t) \) given by Eq. (1), which in turn enters the measured magnetic fields \( B_j(t) \) through Eqs. (3) and (2).

In Fig. 2, we present example plots for the sphere’s trajectory \( z(t) \) [Fig. 2(a)], the signals \( B_1(t) \) [Fig. 2(b)] and \( B_2(t) \) [Fig. 2(c)], and the difference \( \Delta B = B_1 - B_2 \) [Fig. 2(d)] for two values of the viscosity. For generating these plots, we considered two corrections of the simplified description outlined previously.

First, the correction is due to the finite volume of the fluid column, the so-called edge effect. This has been discussed in detail in Refs. 19, 20, and 24, and we here follow the treatment presented therein. In particular, the measured viscosity overestimates the true viscosity because the walls of the fluid’s container effectively push the sphere upward. This is quantified by a correction factor \( K_{\text{edge}} \), which for small Reynolds numbers pertinent to our measurements \( (\text{Re} \lesssim 2) \) is given by

\[ K_{\text{edge}} = \frac{1 + n_5 x^5}{1 + d_1 x + d_3 x^3 + d_5 x^5 + d_6 x^6}, \quad (4) \]

where \( x = 2r/D, \ n_5 = -0.75857, \ d_1 = -2.1050, \ d_3 = 2.0865, \ d_5 = -1.7068, \ d_6 = 0.72603 \). For our case, with \( D = 14 \) mm being the diameter of the cylindrical fluid column and \( r \) being the sphere radius, it is \( K_{\text{edge}} = 1.726 \).

The second correction is due to the fact that the fluxgate sensors are not point sensors, but have a finite volume of a strip geometry with length 2.2 cm, width 1.5 mm, and thickness 0.025 mm. To simulate the sensor signal, we, thus, integrate the magnetic field produced by the sphere in the finite volume of the sensor. The theoretical fits to the data presented here include both aforementioned corrections.

\[ \text{III. EXPERIMENT} \]

To test the magnetic viscometer, we used three viscosity standards, which were oils of known viscosity ranging from about 200 to 3000 cP. The viscosity reference is given by the manufacturer at six different temperatures. We used the reference values at 25°C, but our measurement was not performed exactly at 25°C. Thus, we fitted the temperature dependence of each standard, and from the fits, we found the standards’ viscosity at the actual measurement temperature. In Figs. 3(a)-3(c), we show the temperature dependence of the three viscosity standards, together with the theoretical fits to the functional form \( \log(\eta(T)) = A + B/T + C/T^2 \), suggested in Ref. 40. In the table of Fig. 3(d), we present the nominal values of the viscosity standards at 25°C, along with the corrected values at the actual temperature of our measurement and the corresponding error. As the manufacturer does not quote any errors in the reported

\[ \eta = 500 \text{ cP} \]

\[ \text{FIG. 2. Calculated examples of a falling-sphere trajectory and magnetometer signals for two values of the viscosity, } \eta = 500 \text{ cP (blue curves)} \text{ and } \eta = 550 \text{ cP (red curves). The positions of the two sensors were } a = 4 \text{ cm}, \ b = 5, \ c_1 = 8 \text{ cm for the upper sensor and } c_2 = 5 \text{ cm for the lower sensor. (a) Height of the ball as a function of time. (b) Signal } B_1 \text{ of the upper sensor. (c) Signal } B_2 \text{ of the lower sensor. (d) Difference signal } \Delta B = B_1 - B_2. \text{ Parameter values for this calculation were sphere radius and density } r = 1.46 \text{ mm and } \rho_s = 7.47 \text{ g/cm}^3, \text{ respectively, fluid density } \rho_f = 0.8 \text{ g/cm}^3, \text{ fluid column diameter } D = 14 \text{ mm, and fluid column height } H = 10 \text{ cm.} \]
In each of Figs. 4(a)–4(c), we also display the result for the fit parameter $\eta$, together with the error resulting from the fit. This is calculated by 

$$
\delta(\Delta B)/\sum_{j=1}^{10} \partial f(t_j)/\partial \eta = (\Delta B)^2,
$$

where $\delta(\Delta B) \approx 5 \text{ mV}$ is the measured noise in $\Delta B$, $\Delta B$ are the measured values of $\Delta B$ at time $t_j$, and $\partial f(t_j)/\partial \eta$ is the sensitivity of the theoretical form to $\eta$ at time $t_j$. Incidentally, the quoted intrinsic noise of our fluxgate sensors is $20 \text{ pT/}\sqrt{\text{Hz}}$ at 1 Hz, which within the 1 kHz bandwidth translates to about $\delta(\Delta B) = 100 \mu \text{V}$ noise. Our noise level of 5 mV is mostly due to the sensors operating in the unshielded environment of the lab, without any filters to reduce low-frequency noise. In any case, as will be shown next, the quoted fit-parameter errors stemming from the noise in $\Delta B$ are negligible. Nevertheless, this points to the possibility to obtain, in principle, even lower uncertainties in the estimate of the viscosity, which would take advantage of the intrinsic noise level of the sensors.

The fit errors shown in Figs. 4(a)–4(c) underestimate the precision of our measurements. This is seen by repeating the measurement with the same sample (10 repetitions), in which case, we get standard values, we used as error source a $\pm 0.1^\circ \text{C}$ uncertainty in the temperature of the fluid, leading to an uncertainty in the reference viscosity value around 1%.

In Figs. 4(a)–4(c), we present the actual measurements for the three standard oils, together with the fits to the theoretical form of Eq. (2), including the corrections mentioned in Sec. II. The output voltages of the two fluxgate sensors were digitized with a National Instruments DAQ card at an acquisition rate of 1 kHz. The presented measurements are the differences, $\Delta B$, of the signals recorded by the two fluxgate sensors. An average in time was then performed so that in all cases, the final measured trace for $\Delta B$ has 80 data points. The duration of the measurement was defined by the time when the lower sensor reads a maximum value, increased by 50%. This way, we omit the data points originating from the sphere’s trajectory close to the bottom of the container, in order not to have to include additional corrections due to the finite length of the trajectory.19,20,24

The fits were obtained with the Levenberg–Marquardt algorithm,13 using as fitting parameters the viscosity, the fluid’s density, an amplitude scaling the overall signal, and an additive offset. The positions of the two sensors relative to the fluid column and the initial height of the sphere were measured and kept constant. In particular, $a = 3.3 \text{ cm}$ and $b = 4 \text{ mm}$ define the lateral positions of the center of the sensors, while $c_1 = 8.4 \text{ cm}$ and $c_2 = 5.4 \text{ cm}$ were the height of the upper and lower sensor, respectively. The initial sphere’s height was $H = 9.5 \text{ cm}$. The fitted signal amplitudes were $1.28$, $1.14$, and $1.12 \text{ V}$, corresponding to the samples N100, N350, and N1000. If the sphere has the exact same trajectory with respect to the sensors in every measurement, these three amplitudes should be the same. The amplitude for N100 is 13% larger than the rest, which can be explained by a slightly shifted placement of the current-carrying coil with the sphere (in every trial, we first attach the sphere at the tip of the current-carrying coil and then lower the coil to visually position the sphere just on top of the fluid surface).

As seen in Figs. 4(a)–4(c), there is excellent agreement of the theoretical fits with the measured data. The slight discrepancy between data and fits observed in the beginning phase of the signals is conceivably due to an interplay of effects not taken into account in our theoretical model and related to the splash of the sphere on the liquid surface,24 to surface tension, and wetting.

### IV. RESULTS AND ERROR ANALYSIS

![Table 1](https://example.com/Table1.png)

<table>
<thead>
<tr>
<th>Fluid Sample</th>
<th>Nominal Temperature ($^\circ \text{C}$)</th>
<th>Nominal Viscosity (cP)</th>
<th>Actual Temperature ($^\circ \text{C}$)</th>
<th>Corrected Viscosity (cP)</th>
</tr>
</thead>
<tbody>
<tr>
<td>N100</td>
<td>25</td>
<td>215.3</td>
<td>25.4</td>
<td>210 ± 2</td>
</tr>
<tr>
<td>N350</td>
<td>25</td>
<td>707.5</td>
<td>26.0</td>
<td>730 ± 7</td>
</tr>
<tr>
<td>N1000</td>
<td>25</td>
<td>3008</td>
<td>26.0</td>
<td>2750 ± 30</td>
</tr>
</tbody>
</table>

FIG. 3. Temperature dependence of the viscosity, $\eta(T)$, as given by the manufacturer38 at six different temperatures for three reference standard oils: (a) N100, (b) N350, and (c) N1000. The solid line is a fit to the functional form $\log(\eta(T)) = A + B/T + C/T^2$, with the respective fit parameters $A$, $B$, and $C$ shown in the insets. In the formula, $T$ is the absolute temperature. We use the fit to correct for the standard viscosity since our measurements were performed around 25 $^\circ \text{C}$, but not exactly at 25 $^\circ \text{C}$, which is the second temperature data point provided by the manufacturer. The fit-parameter errors are negligible. (d) The table shows the nominal viscosity reference values at 25 $^\circ \text{C}$ and the actual values calculated from the fit at the actual temperature of our measurements. The errors quoted in the values of the corrected viscosities in the last column of the table derive from an uncertainty of 0.1 $^\circ \text{C}$ in the actual temperature shown in the fourth column of the table.
a relative standard deviation in the viscosity estimates around 3%. This is the final quoted measurement error in the table of Fig. 4(d), which shows how the measured viscosities compare with the corresponding standard values, demonstrating a very good agreement given the simplicity of our setup. Sources of the 3% variability could be short-term temperature drifts or small rotations of the sphere due to small density inhomogeneities (small bubbles) of the fluid.

Finally, as noted in Sec. I, one practical advantage of our methodology is that it allows one to directly access the dynamic viscosity \( \eta \), in contrast to conventional falling-sphere viscometers requiring knowledge of the fluid’s density in order to extract the dynamic viscosity from the measurement of the terminal velocity. While we do leave the fluid density \( \rho_f \) as a fitting parameter in the fitting algorithm, the data reported herein do not provide for a precise measurement of \( \rho_f \). This is seen qualitatively by the fact that \( \rho_f \) enters Eq. (1) through the expression \( \frac{1}{C_0 \rho_f} = \rho_s \), and the ratio \( \rho_f/\rho_s \) is about 0.1; hence, the fluid density only mildly affects the sphere’s trajectory. Quantitatively, it becomes evident that the \( \chi^2 \) dependence on \( \rho_f \), where \( \chi^2 = \sum_{t=1}^{80} \left( f(t) - (\Delta B) \right)^2 \), has a very shallow minimum. In particular, \( \chi^2 \) is about two orders of magnitude less sensitive on \( \rho_f \) than it is on \( \eta \). The result is that the fitted values for the fluid’s mass density indeed follow the trend of the numbers reported by the manufacturer for the three standard fluids (N100: 0.874 g/cm³, N350: 0.891 g/cm³, and N1000: 0.921 g/cm³) but are about 5%–20% off. Moreover, the exact discrepancy depends on the particular parameter update step chosen in the fitting algorithm. Such variability in the density estimate translates into viscosity estimate changes within the 3% error quoted above.

V. DISCUSSION OF MEASUREMENT UNCERTAINTIES

Since this viscometer is based on measuring the magnetic field produced by the magnetized sphere, it should be operated away from ambient magnetic-field sources, in particular, strong magnets that might saturate the sensors or affect the sphere. If this is not an issue, it could be time-dependent magnetic fields different at the two sensors that could cause extra noise since the difference signal removes common mode noise, while a signal difference constant in time is taken care of by the fitted background of the difference signal \( \Delta B \).

Regarding possible forces on the magnetic sphere, the ferromagnetic core of the fluxgate sensors themselves produces a magnetic field; thus, we placed the sensors at a horizontal distance of 3.3 cm from the fluid sample. We measured the magnetic gradient produced by the sensors at the position of the fluid, and it was found to be around 1 mG/cm. Taking into account the remanence of the sphere (1.3 T), we estimate the sphere’s magnetic moment and find that the force on the sphere due to this gradient is four orders of magnitude smaller than the sphere’s weight. Hence, the sensors themselves do not affect the sphere. In any case, if this viscometer is required to operate close to strong laboratory magnets, it should be enclosed in a magnetic shield.

Regarding a possible rotation of the sphere upon release from the current-carrying coil, if there was such a rotation, the theoretical model would not be able to fit the data since the theoretical model assumes constant magnetization of the sphere along the...
z axis (the sphere’s trajectory). Nevertheless, we also used a magnetized cylinder and visually inspected the fall, which did not exhibit any noticeable rotation upon release.

Another concern, due to the sensitive temperature dependence of viscosity, could be heating of the fluid sample by the frictional Stokes force. With an-order-of-magnitude calculation, it is seen that such an effect should be negligible. Indeed, for a fluid specific heat on the order of 1 J/g K and setting the work done by the sphere’s weight (equal to the opposing forces when in equilibrium) equal to the heat transferred to the fluid, we find a temperature change on the order of μK.

Yet another concern could be the fluid’s density fluctuations, possibly causing random rotations of the magnetized sphere and secondly causing small random deviations from the trajectory of Eq. (1) due to a random change in the buoyant force. However, thermodynamic density fluctuations resulting from particle number fluctuations, on the order of 1/√N where N is the number of fluid particles in the macroscopic volume occupied by the sphere, are negligible since N ≈ 10^25. On the other hand, there could be density fluctuations due to more rudimentary issues, such as tiny bubbles in the fluid. These, however, are much harder to systematically quantify.

Finally, we checked the sphericity of the spheres and found the non-sphericity to be at the level of 0.5%, which translates into 1% uncertainty in the parameter τ, due to the r^2-dependence of τ. This error, along with other uncertainties considered in Refs. 19 and 20, is negligible with respect to the precision of 3%, which also reflects the accuracy of this measurement. In summary, in this work, our aim is not to compete with previous realizations of falling-sphere viscometers in terms of precision/accuracy, but to introduce a new kind of falling-sphere viscometer, the precision and accuracy of which we hope to improve in future refinements of the method.

VI. CONCLUSIONS

We have presented a simple falling-sphere viscometer using a magnetic sphere and two fluxgate sensors continuously monitoring the sphere’s fall within the test fluid. The viscometer’s precision could be further improved by modifying the design details of this methodology, in particular, the temperature stability. The fluid volume used in this work is 15 ml, and it can be further reduced by using a smaller diameter sphere and a smaller fluid container. The method can also work at higher temperatures, at least up to 80°C quoted by the magnetized sphere manufacturer, which is a fraction of neodymium’s Curie point. One could even conceive significant miniaturization of this technique toward measuring ultra-low fluid sample volumes by using different kinds of magnetometers, such as diamond sensors, or miniaturized atomic magnetometers.

ACKNOWLEDGMENTS

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AUTHOR DECLARATIONS

Conflict of Interest

The authors have no conflicts to disclose.

Author Contributions

C. Patramanis-Thalassinakis: Investigation (equal). P. S. Karavelas: Investigation (supporting). I. K. Kominis: Conceptualization (lead); Formal analysis (lead); Funding acquisition (lead); Project administration (lead); Supervision (lead); Writing – original draft (lead).

DATA AVAILABILITY

The data that support the findings of this study are available from the corresponding author upon reasonable request.

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The user of this methodology should consult the MSDS of the fluids under test in order to gauge their compatibility with neodymium or any other material the falling sphere is made of.

See https://www.magnetandel.de/neodymium-magnet-spheres for “Magnet Spheres.”


See https://psl-rheotek.com for information on the oil standards used in this work.


F. Strat, Data Fitting and Uncertainty (Vieweg Teubner Verlag, 2016).


